ing abutments, and connected with means for drawing in one chain and loosening the other simultaneously, the spring being suitably connected with the bolt E² having semispherical bearing E¹³, rotating against the curved bearings in the saddle or projection, the recess in said saddle through which the bolt passes being enlarged rearwardly, substantially as and for the purpose specified. 27th. In a traction engine having front axle, capable of being turned on a pivot connected to the frame of the machine, and chains connected respectively to the outer or end portions of the axle and converging rearwardly against the deflecting abutments, and connected with means for drawing in one chain and loosening the other simultaneously, the spring being suitably connected with the bolt E², taneously, the spring being suitably connected with the bolt E', having buttons or semispherical bearing E''s, rotating against the curved bearings in the axle saddle or projection, the recess in said saddle through which the bolt passes being enlarged rearwardly, the bolt E''s, being provided with the head E''s, and the cylinder E's, the said head lying within the cylinder E's, and against the head E's, of the latter, and the spring E's, within the cylinder, and held in those by these E's ward the deep best E's whole E' by, of the latter, and the spring E^* , within the cylinder and held in place by the cap E^* , and the draw bolt E^* , whose head E^* , is within the cylinder and against that end of the spring which is opposite to the one which bears against the cap E^* , the bolt E^* passing through the cap being provided with the hook E^* , for connection with the chain, substantially as and for the purposes specified. 28th. In a traction engine having front axle, capable of being turned on a pivot, connected to the frame of the machine, and chains connected respectively to the outer or end portions of the axle and converging rearwardly against the deflecting abutments, and connected with means for drawing in one chain and loosening the other simultaneously, the spring being suitably connected with the bolt E2, having semispherical bearing E13, rotating against the curved bearings in the axle saddle or projection, substantially as curved bearings in the axie saddle or projection, substantianly as and for the purposes specified. 29th. In a traction engine having front axle, capable of being turned on a pivot connected to the frame of the machine, and chains connected respectively to the outer or end portions of the axle and converging rearwardly against the deflecting abutments, and connected with means for drawing in one chain and loosening the other simultaneously, the spring being suitably connected with the bolt E^2 , having semispherical bearing E13, rotating against the curved bearings in the axle saddle or projection, the recess in said saddle through which the bolt passes being enlarged rearwardly, substantially as and for the purposes specified. 30th. In a traction engine, the frame for supporting the machine and constituting the reaches between the front head or cross block and the rear axle, consisting of piping Λ^2 , Λ^2 , forming the conduit, the piping at one end of the machine being connected by the conduit A^{3} , and the other end of the machine, one of the pipes A2, being connected with the pipe A6, connected in turn to the space between the jacket of the engine and the cylinder thereof, the adjacent end of the other pipe being connected by the auxiliary pipe A⁴, to the pump, substantially as and for the purposes specified. 31st. In an engine, the crank or driving shaft F, carrying the expander \mathbf{F}^2 , fixed thereon, the expander having the opening \mathbf{F}^n , in its outer portion, and the friction shell \mathbf{F}^n , surrounding the expander having the opening \mathbf{F}^n , in der and connected to the driving gear F4, the shell and gear mounted loosely on the shaft, and the beveled expander key F⁶, and expander lever F⁷, arranged to be moved outwardly from the shaft by key F⁸, and still farther separate the portions of the expander opposing each other at the slit or opening F⁸, thereby increasing the diameter of the expander, substantially as and for the purposes specified. 32nd. In an engine, the crank or driving shaft F, the driving gear F*, provided with the shell F*, turning loosely on the shaft, and an expander F^2 , fixed to the shaft and provided with a lever pivoted substantially at F^8 , in connection with the slot F^a , of the expander for enabling the movements of the lever to increase the size of the expander, the free end of the lever necrease the size of the expander, the free end of the lever carrying the adjustable set screw and reciprocating under and in contact with the set screw, and means for reciprocating the key, substantially as and for the purposes specified. 33rd. In an engine, the crank or driving shaft F, the driving gear F⁴, provided with the shell F³, turning loosely on the shell, and an expander F², fixed to the shaft and provided with a lever pivoted at F³, and in connection with the slot F³, of the expander expander, the free end of the lever carrying the adjustable set screw F¹⁰ and expander key V⁶ in contact with for enabling the movements of the lever to increase the size of the \mathbf{F}^{10} and expander key \mathbf{F}^{6} , in contact with the set screw and sliding in the groove \mathbf{F}^{25} in the shaft and means for reciprocating the expander key, substantially as and for the purposes specified. 34th. In an engine, the crank or driving shaft F, the driving gear F⁴, provided with the shell F³, turning loosely on the shaft, and an expander F², fixed to the shaft and provided with a lever pivoted substantially at F⁸, in connection with the slot F⁹, of the expander for enabling the movements of the lever to increase the size of the expander, the free end of the lever carrying the adjustable set screw expander, the free end of the level carrying the adjustance set series \mathbb{R}^{10} and expander key \mathbb{R}^{6} , in contact with the set screw and sliding in the groove \mathbb{R}^{25} in the shaft, and means for reciprocating the expander key, and the shifter sleeve \mathbb{R}^{5} , provided with abutments \mathbb{R}^{13} and the forked lever \mathbb{R}^{12} , operating between said abutments and pivoted at \mathbb{R}^{14} , to the bracket \mathbb{R}^{20} to the engine, substantially as and for the purposes specified. 35th. In an engine, the combination of the driving shaft F and driving gear F⁴, pinions G and G² meshing with the spur wheel G³, mounted on the clutch gear shaft G⁴, the said pinions and gear being located in the frame G⁵, G⁵, escillating on the said shaft G⁴, and the nut G⁹ located in said frame

and screw G^{*}, passing through and engaging the nut G^{*} and journaled in the bracket G^{*}, substantially as and for the purposes naled in the bracket G¹³, substantially as and for the purposes specified. 36th. In an engine, the combination of the driving shaft F and driving gear F⁴, pinions G and G², meshing with the spur wheel G³, mounted on the clutch gear shaft G⁴, the said pinions and gear being located in the frame G⁵, G⁵, in the frame G⁵ G⁵, said frame oscillating on the said shaft G⁴, and the nut G⁹, located in said frame, and the screw G³, passing through and engaging the nut G³ and journaled in the bracket G¹³, set screw rod being provided with the collar G¹⁴, provided with the notch G¹⁹ and the lever G^{16} , pivoted at G^{17} , having the latch G^{18} , for engaging said notch G^{19} , and tail piece G^{20} and having the elastic spring set G^{21} , having G13, and tail piece G23 and having the elastic spring set G21, having a point provided with two beveled sides engaging the point G23, on the bracket, which latter point also has two beveled sides, screw shaft having nut or detent G13, preventing it from riding out of the bracket G13, substantially as and for the purposes specified. 37th. In combination with the oscillating frame G5, G5, pivoted on a shaft G4, the spur gear G3 and pinions G2, and driving gear F4, mounted on its separate shaft, the screw G3, working in the notch G3 in one portion of the oscillating frame G5, G5, and mechanism for preventing the rotation of the lod except by human agency, to wit, lever G13, pivoted at G17, and having latch G18, for engaging the notch G13 of the screw and tail piece G23, spring set G21, pressed down by means of the spring G27, located in the chamber G20 of the lever, and the spring set being provided with the nut or detent G23 and at its free or operating end being alternately provided with two bevels for engagement with two bevels on the latch vided with two bevels for engagement with two bevels on the latch point of the bracket, G13, substantially as and for the purposes specified. 38th. In combination with the oscillatory frame G5, G5, carrying the gears in combination with the driving gear, the screw G* arranged for oscillating the frame G5, and passing through the bracket G12 and provided above the bracket with collar G14 and below the bracket with the nut or detent G15, the passage through the bracket being enlarged from the center in both directions to allow the necessary oscillation of the groove as frame G5, G5, oscillates, substantially as and for the purposes specified. 39th. In combination, the driving wheels, a shaft 1 for operating one of said driving wheels, and a sleeve 12 for operating the other of said wheels. the shaft being received into said sleeve, and the eye 13 located in the recess 14 of said sleeve, and surrounding the shaft I, the eye having a bolt 15 and a cap 17 over recess 14, the bolt 15 passing through the cap and nut 16 thereof, substantially as and for the purposes specified. 40th. In the mechanism for enabling the driving wheels to rotate the one faster than the other, while both are positively driven, the combination of the driving wheels shaft I supported near one end in journal bearings B* and pinion I20 fixed on said shaft I and internal gear BD on the adjacent driving wheel B17, shaft I and internal gear BD on the adjacent driving wheel B¹¹, sleeve 12 loosely receiving the other end of the shaft and journaled in a bearing as B⁸, a pinion I⁸, fixed on said sleeve and engaging internal gear BD, and said internal gear located on the adjacent driving wheel B¹⁷, a driving wheel positively driven by the engine and engaging a pinion I¹⁰ or I¹⁰, said pinion being journaled on shaft I, and be velled pinions I¹² on opposite sides of hub of pinion I¹⁰ revelving leasely on shaft I, and the adjacent baseled gear I¹⁰ I'' revolving loosely on shaft I, and the adjacent beveled gear I' and I16 respectively engaging said pinion I12 on opposite sides thereof, one of these beveled gears being fixed to the shaft I and the other to the sleeve I², the sleeve I² having a recess I⁴, an eye I³ surrounding the shaft, and a bolt I⁵ connected to the eye and means for drawing the eye against the shaft I, substantially as and for the purposes specified. 41st. In the mechanism for stopping the K, fixed on shaft I, operating the driving wheels B¹⁷, through suitable intermediate mechanism, and the brake band K⁴, K⁵, attached at one end to the frame of the machine, and at the other end at K⁷, to the nut K³, nut K¹³, and screw sleeve K¹⁶, and screw rod K15, screwing through the nut K13, secured in the sleeve by the collars K17, substantially as and for the purposes specified. 42nd. In the mechanism for stopping the machine, the brake wheel K. fixed on shaft I, operating the driving wheels B17, through intermediate mechanism, and the brake band K4, K5, attached at one end to the frame of the machine and at the other end at K⁷, to the nut K¹³, nut K¹³, and screw sleeve K¹⁶, and screw rod K¹⁸, screwnut K¹⁷, nut K¹⁷, and screw sleeve K¹⁸, and screw rod K¹⁸, screwing through the nut K¹³, secured in the sleeve by the adjustable collars K¹⁷, substantially as and for the purpose specified. 43rd. In the mechanism for stopping the machine, the break wheel K, fixed on shaft I, operating the driving wheels B¹⁷, through interfixed on shaft I, operating the driving wheels B¹⁷, through intermediate mechanism, and the brake band K⁴, K⁵, attached at one end to the flange B³⁴, of the axle cap, and at the other end at K⁷, to the nut K¹⁵, nut K¹⁵, and screw sleeve K¹⁶, and screw rod K¹⁵, screwing through the nut K¹³, secured in the sleeve by the collars K¹⁷, substantially as and for the purposes specified. 44th. The combination of the shaft M, and the pulley M⁶, friction shell connected to gear wheel meshing with gear on the crank shaft F, and an expander and expander key fixed to shaft M, and device for shifting the expander key, substantially as and for the purpose specified. 45th. In combination with the piping A², A⁶, A², and A⁴, the caps P², P², on said piping A², A², and the buffer P, whose ends are connected to the said caps, substantially as and for the purposes specified. 46th. In combination with the piping A², A⁶, A⁵, poses specified. 46th. In combination with the piping A2, A6 poses specified. 46th. In combination with the piping A^* , A^* , and A^* , the caps P^* , P^* , on said piping A^* , A^* , and the buffer P, whose ends are connected to the said caps, and whose centre is provided with the concave piece P^* , centrally located therein, the buffer being of a box shape, substantially as and for the purposes specified.