rib shortening, the live load being too small to materially change the stress ratios or increase the fibre stresses. Stresses were computed at the critical points along the ring and a tabulated comparison given in Table I.

A study of the table, together with the stress sheets of Fig. 1, will reveal the following significant facts:—

- (1) Arch ring "B" contains about 21 per cent. more material than arch ring "A," thus increasing the cost for materials and for mixing and placing of concrete in about that ratio.
- (2) Notwithstanding the above increase in material the average compression in arch ring "B" is about 11½ per cent. and the average tension about 90 per cent. in excess of the corresponding values in arch ring "A" for the same loadings.

(3) The deflection under dead load stress at the crown is .0468 in. for arch ring "A" and .037 in. for arch ring "B." Both these movements being comparatively small, the increased rigidity of arch ring "B" is but a small advantage.

(4) The abutment toe pressure for arch ring "B" is 14 per cent. in excess of that for arch ring "A" due to the greater value and less inclination of thrust at the springing lines.

$$\left[\int_{0}^{\frac{Z}{2}} Y \frac{ds}{I}\right] \div \left[\int_{0}^{\frac{Z}{2}} \frac{ds}{I}\right] \text{ where}$$

"ds" is an infinitesimal increment of axis length, "I" the moment of inertia at any point, and "y" the ordinate to the same measured from a straight horizontal line through the crown.

It is thus seen that the comparison, both as regards cost and stress condition, is favorable to arch ring "A" wherein the depth of section is not materially increased for the central half of the span. It will be noted that the equilibrium polygon for temperature is a straight hori-

zontal line whose distance below the crown is a fixed quantity for a given arch.¹ Obviously, then, the temperature stress will decrease from crown towards spring line, passing through a zero value at the intersection of the equilibrium polygon with the arch axis and assuming its original or crown value (with opposite sign) at a point symmetrical with the crown with reference to the equilibrium polygon as an axis of symmetry. This latter point may be fermed the "equivalent crown point for temperature stress." It is thus apparent that as a provision against temperature no increase in section is needed between the true and equivalent crown points.

Inasmuch as the equilibrium polygon can, by a proper selection of the central portion of the ring, be made to follow the arch axis between these two points, the dead load bending stress may be practically eliminated. The only remaining loading to be taken care of is the live and it can easily be shown that for highway loadings the increase in live load moments between the true and equivalent crown points is so small as to be negligible.

Based not only upon the example given herewith, but also upon the results of a large number of arch rings analyzed in the offices of the Iowa State Highway Commission, we are warranted in the conclusion that—

The economical section for short-span highway arch structures of which the span under discussion is typical is obtained when the axial curve is so chosen that the equilibrium polygon for dead loading is practically coincident therewith and when the depth of section between the true and equivalent temperature crown point as defined above is not materially increased.

¹Note.—This distance, as can easily be shown mathematically, is dependent upon the elastic properties of the ring, and is measured by the quantity.

*Tension in

*Compres-

	Loading.	Mor	ment—	Th	re	
(D .		Ring A.	Ring B.	Ring A.	Ring B.	re Ring

Table I.—Comparative Stress Values in Arch Rings A and B.

Section.	Loading.	Ring A. Ring B.		Ring A. Ring B.		reinforcing steel. Ring A. Ring B.		Ring A. Ring B.	
Crown {	Dead load Temperature rise Rib shortening	-2,826 $-5,540$ $+1,385$	- 2,416 - 8,108 + 1,824	24,000 3,600 — 900	25,370 8,123 — 1,828	King A.	King B.	King A.	King 2
	Total	- 6,981	- 8,700	26,700	31,665	785	1,300	418	521
Section 10 {	Dead load Temperature rise Rib shortening	+ 2,430 + 6,260 — 1,565	+ 3,496 + 9,292 — 2,095	25,500 3,400 — 850	26,220 7,900 — 1,782				
	Total	+ 7,125	+ 10,693	28,050	32,338	112	401	373	371
Section B. B. 3	Dead load Temperature drop Rib shortening	- 7,710	-12,616	- 3,340	26,200 — 7,820 — 1,760				
	Total					2,750	6,070	396	386
Point 11 {	Dead load Temperature drop Rib shortening	- 4,711 -13,080 - 3,270	-2,476 $-29,292$ $-6,590$	27,750 — 3,270 — 818	27,650 — 7,560 — 1,700				
	Total	-21,061		23,662	18,390	9,050	23,000	506	655
Spring line {	Dead load Temperature drop Rib shortening								
	Total	-50,286	-90,706	25,450	21,760	19,860	31,000	603	631

*The stresses in the material were figured on the basis of a "cracked section"—that is, the steel assumed to take the entire tensile stress. In reality such a cracked condition would alter the position of the neutral axis and therefore the pressure curve. However, for our present purpose this method gives a very good indication of the actual comparative stress conditions.